

Aerodynamics & blade technology II Vortex-Induced Vibrations

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General context

- Cross-sectional activities within Wind Turbine Design division of DTU
- Aeroelasticity modelling for rotor design
- Involving several researchers and disciplines.
 - N. N. Sørensen, F. Zahle, C. Grinderslev (Computational Fluid Dynamics and Fluid Structure Interaction)
 - > T. Barlas (multi-body analysis and experiments)
 - > A. Li, G. Pirrung (engineering aerodynamic models)
 - > N. Ramos-Garcia (vortex methods)
 - ▶ ...
- Two-folded mission:
 - > Progress in the **understanding** of **particular** aeroelastic **phenomena**
 - Improve current tools (high fidelity -> engineering models)

Drivers

> The **times** we live in

- Current and next generation of wind turbines
- Rotor-upscaling: as a shifter of the role played by aeroelasticity
- > The pursuit of **innovative** designs and concepts, examples:

Low wind project

- Smart Tip project
- Working together with industry, to answer some questions around numerical methods:
 - > What could we do **better**?
 - What could we do faster?
 - What are we missing?

https://windenergy.dtu.dk/english/research/researc h-projects/lowwind



https://windenergy.dtu.dk/english/research/research-projects/completed-projects/smart-tip

DTU coupling: the Trojan Horse

- Combines different solvers of the department
- Unified aeroelasticity framework



Focus of this presentation

- > A phenomenon we are probably missing
 - Vortex Induced Vibrations (VIV)
- Occurring at:
 - Blades (standstill)
 - Towers
- Could limit design and impose operational constraints



Experiment at University of Southampton [https://www.youtube.com/watch?v=UHFJPmtKqGo]



Introduction

Methods

- Applications
- Conclusions



About the numerical methods

- > Vortex shedding is hard to model with engineering models
 - > Usually requires of the so-called **Computational Fluid Dynamics** (CFD) tools
- > VIV are hard to model in a decoupled way:
 - > Usually requires to pass to a Fluid Structure Interaction (FSI) approach
 - > Involving not only a CFD solver, but also a structural model
- > For the DTU case, we follow a staggered approach:
 - Existing CFD solver (EllipSys3D), in-house commercial software
 - Existing FEM-multibody solver (HAWC2), in-house commercial software



Example of numerical model

> Model of the IEA 10MW blade [Horcas et al. (2020)]

- Body-fitted structured mesh of 12.6 millions of cells
- > Hybrid Reynolds-averaged Navier Stokes (RANS) and large eddy simulation (LES)



> Typical CPU cost per simulation, modern cluster: **12-24 hours** when using **300 processors**



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WT edgewise vibration in Standstill

- > Can occur at **low-to-moderate** wind **speeds** [10-20] m/s, exciting **first edgewise**
- > The phenomenon seems to appear when:
 - Incoming flow ≈ normal to blade
 - AOA of sections around ±90 deg
 - High inclination angles I (> 20 deg)
 - Velocity component from tip to root.

- In operation, it means:
 - Installation/Yaw failure
 - Blade braked at a certain angle.



Main challenges for blades VIV

- > Complexity of the flow: requiring high fidelity (CPU cost)
- > Variability of **inflow conditions** (and interference with other wakes).
- > Dependence on **blade geometry** and **structural properties**.
- > Hard to relate to traditional [circular cylinder] VIV research. But we anticipate:



Cross-section + Re for VIV



Ding et al. (2015)





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Example of VIV animation



Sensitivities of WT blade VIV

> The region of VIV is very sensitive to the local blade geometry at the tip.



> Installation of trailing edge flaps/spoilers may also help to prevent the phenomenon.



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Tower vibrations

> Some aspects regarding **tower VIV** may **simplify** the problem:

- Less complex geometry (closer to literature)
- More normal flow
- > Experience on **similar structures** (e.g. chimneys)



https://www.youtube.com/watch?v=rlpUhgfEZPU

However:

- High Reynolds literature is still very scarce
- Particularities of geometry and structure
- WT assembly topology and transportation



Figure 14. Instantaneous non-dimensional vorticity magnitude for $U^* = 3.1$ at the maximum positive displacement: $t/T_s = 143.5$ (left) and $t/T_s = 146.5$ (right). The vorticity is filtered to the range $0 \le \xi D/U_{\infty} \le 5$. Figures taken from Derksen (2019).

Vire et. al (2020)



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Conclusions

Current and future WT designs could be subjected to:

- Blade VIV in standstill
- Tower VIV
- High fidelity models are needed
 - > Requiring of high **CPU time**
 - Requiring of a muti-displicinary team to be successfully run
- > So far, numerical simulations have predicted the phenomena
 - Validation efforts are needed
- > Industry and academia should work together to solve such a complex problem
 - Through commercial agreements
 - Research-oriented projects



End of presentation



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